

AESB 2320 Part 2 Exam 19 April 2017

1. How does this differ from section 9.2 of BSL1?

- different B.C. at $r=R$ (Newton's law of cooling)
- non-uniform generation function $Se(r)$.

The B.C. does not affect the shell balance. Newton's law of cooling does not interact with the control volume between r and $(r+\Delta r)$. It doesn't enter the derivation until Eq. 9.2-12. The non-uniform Se , however, alters the integration of Eq. 9.2-6 to 9.2-7. Eq. 9.2-7 assumes Se is a constant. Therefore the last equation that applies is Eq. 9.2-6.

(This same issue came up in the Part 1 exam (15 March), problem 1. There, the nonuniform viscosity μ meant that one could not integrate for q_T assuming it is constant.)

2. a) Newton's law of cooling specifies $q = h(T - T_a)$

i) A macroscopic energy balance on the cylinder:

rate of heat lost to surroundings: $(2\pi RL)h(T - T_a)$

no convection: $(TR^2L)\rho CP \frac{dT}{dt}$

no generation, no convection

$$-(2\pi RL + 2\pi R^2)h(T - T_a) = \pi R^2 L \rho CP \frac{dT}{dt}$$

$$\frac{d(T - T_a)}{dt} = -\frac{2\pi RL h}{\pi R^2 L \rho CP}$$

$$= -\frac{2 h}{R \rho CP}$$

$$\frac{d(T - T_a)}{T - T_a} = -\frac{2 h}{R \rho CP} +$$

$$\text{L}, (T - T_a) = -\frac{2 h}{R \rho CP} t + C_1$$

$$\text{at } t=0, L, (T - T_a) = 200 - 20$$

$$\frac{T - 20}{200 - 20} = \exp\left(-\frac{2 h}{R \rho CP} t\right) = \exp\left(\frac{-2.157}{(0.025)(1000)(30)} t\right)$$

$$= \exp(-2.79 \cdot 10^{-3} t)$$

c) There was some confusion among students on the meaning of "gradient," which means derivative with respect to position ($\partial T / \partial r$), not time ($\partial T / \partial t$). As it happens, the maximum in both $(\partial T / \partial r)$ and $(\partial T / \partial t)$ occurs at $t=0$.

From Fourier's law in the solid,

$$q_r = -k(\partial T / \partial r)$$

Since the heat flux at the surface is given by

Newton's law of cooling, then at the surface

$$-k \frac{\partial T}{\partial r} = h(T - T_\infty)$$

The maximum in $(T - T_\infty)$ occurs at $t=0$; it decreases with time.

$$\left. \frac{\partial T}{\partial r} \right|_{r=R, t=0} = -\frac{q_r}{k} = -\frac{h(T - T_\infty)}{k} = -\frac{h(180)}{k} = -\frac{50(180)}{35} = 252 \frac{\text{°K}}{\text{m}}$$

3. Convective heat transfer is in series with internal conduction.

For processes in series, the slow step controls; therefore we want internal conduction to be as fast as possible compared to convective heat transfer to the surface.

Internal conduction depends on $\frac{dt}{R^2}$, with $d = \frac{k}{\rho c_p}$.

For convective heat transfer, leading to a macroscopic energy balance like that in problem 2(b), temperature scales with $\frac{ht}{\rho c_p}$ (see problem 2). Both processes slow down as ρc_p increases, but a larger k makes internal conduction go faster. Therefore the best metal has the larger k . Metal A has $10 \times$ larger k ; it is the better choice, even though (ρc_p) is larger.

[When I first wrote this problem, I thought the only thing that matters is d . I also accepted answers specifying metal A, because d is about $6x$ larger, though the complete answer is more complicated.]

4. Because of the perfectly insulated surface, this problem is equivalent to a cube, 0.2 m on a side, where we calculate T at center.

$$a = \frac{k}{\rho c_p} = \frac{23}{7820 \cdot 4181} = 6.38 \times 10^{-6}$$

$$\text{At } 5 \text{ min}, \frac{xt}{b^2} = \frac{6.38 \cdot 10^6 (3.00)}{(0.1)^2} = 0.19$$

From chart for slab, $\frac{T_1 - T}{T_1 - T_0} = 0.77$ at $\frac{xt}{b^2} = 0$

for cube, " $= (0.77)^3 = 0.496$

$$\frac{100 - T}{100 - 0} = 0.496; T = 54^\circ\text{C}$$

$$\text{at } 8 \text{ min}, \frac{xt}{b^2} = \frac{6.38 \cdot 10^6 (4.80)}{(0.1)^2} = 0.306$$

From chart for slab, $\frac{T_1 - T}{T_1 - T_0} = 0.6$ at $4.8/b = 0$

for cube, " $= (0.6)^3 = 0.29$

$$\frac{T_1 - T}{T_1 - T_0} = 0.29 = \frac{100 - T}{100 - 0} \Rightarrow T = 0.71^\circ\text{C}$$

5. For this problem, we need $\frac{T - T_0}{T_1 - T_0}$. We already solved for $\frac{T_1 - T}{T_1 - T_0}$ in problem 4.

$$\text{At } 8 \text{ min}, \frac{T - T_0}{T_1 - T_0} = \frac{71 - 0}{100 - 0} = 0.71$$

$$5 \text{ min}, " \frac{54^\circ\text{C}}{100 - 0} = 0.54$$

Dimensionless T is $\frac{T - 0}{100 - 0}$. Second change is

opposite to first

$$\frac{T - T_0}{T_1 - T_0} = \left(\frac{T - T_0}{T_1 - T_0} \right)_{8 \text{ min}} - \left(\frac{T - T_0}{T_1 - T_0} \right)_{5 \text{ min}} = 0.71 - 0.54 = 0.17$$

$$\frac{T - 0}{100 - 0} = 0.17; T = 17^\circ\text{C}$$